

Now $\Delta V = -A\delta$ where A is the area of the piston ($V = AL$). So

$$\Delta S = \frac{3}{2} k \frac{f\delta}{E/N} - \frac{NkA\delta}{AL}$$

Now $\frac{E}{N} = \frac{3}{2} kT$, so

$$\Delta S = \frac{f\delta}{T} - \frac{Nk\delta}{L} = \frac{\delta}{T} \left(f - \frac{NkT}{L} \right)$$

The entropy increases until equilibrium is reached at $L = L_0$ where $\Delta S = 0$

$$0 = \Delta S = \frac{\delta}{T} \left(f - \frac{NkT}{L_0} \right)$$

$$\text{So } f = \frac{NkT}{L_0} \text{ and } L_0 = \frac{NkT}{f}$$

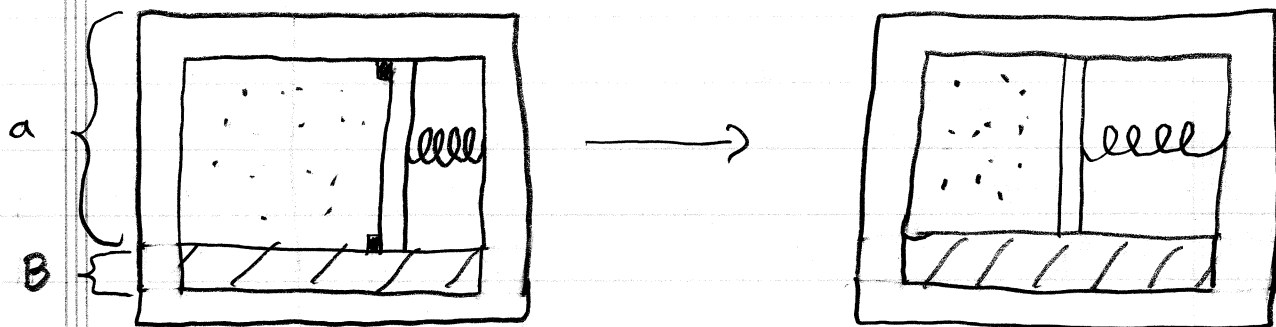
Setting $f = pA$ where p is the pressure,

$$\text{we get } pL_0A = pV = NkT$$

which is the ideal-gas law.

When a constraint is removed, the number Ω of microstates rises, and so does the entropy $S = k \ln \Omega$.

This "irreversibility" is due to the choice of a constrained initial state, e.g., all molecules on one side of the box, and not to a breakdown of time-reversal symmetry.



System a (gas & spring) can exchange heat with system B, which is a big slab of iron whose temperature T_B remains nearly constant. B is a thermal reservoir.

We release the brakes, and the spring compresses the gas whose temperature T_a initially rises but then subsides to $T_B = T$ as the combined system reaches equilibrium.

So $E_{\text{gas}} = \frac{3}{2} NkT$ does not change, but

E_{spring} does change — it goes down. So

$$\Delta E_a = \Delta (E_{\text{gas}} + E_{\text{spring}}) = -f\delta. \quad \text{So}$$

$\Delta S_a = \Delta S_{\text{gas}} + \Delta S_{\text{spring}}$, we ignore ΔS_{spring} .

$$\text{So } \Delta S_a = \Delta S_{\text{gas}} = \Delta k \ln \frac{E_{\text{gas}}^{\frac{3N}{2}} V^N}{\Delta} = \frac{Nk \Delta V}{V}$$

$$\Delta V = -A\delta, \quad \text{so}$$

$$\Delta S_a = - \frac{NkAS}{AL} = - \frac{Nk\delta}{L}.$$

We must include system B. Now

$$T = \left(\frac{dS_B}{dE_B} \right)^{-1} \quad \text{so} \quad \Delta S_B = \frac{\Delta E_B}{T} = - \frac{\Delta E_a}{T}.$$

Thus when $S = S_a + S_b$ is maximal

$$\begin{aligned}\Delta S &= \Delta S_a + \Delta S_b \\ &= -\frac{Nk\delta}{L} - \frac{(-f\delta)}{T} \\ &= \frac{\delta}{T} \left(f - \frac{NkT}{L} \right)\end{aligned}$$

which again gives $L_0 = \frac{NkT}{f}$ and

with $f = pA$

$$pV = NkT$$

which is the ideal-gas law. (that was 6C)

So when we consider a small system "a" in thermal contact at fixed volume with a much larger thermal reservoir, the release of a constraint minimizes

$$F_a = E_a - TS_a.$$

Here T is the fixed temperature of the thermal reservoir. $F = E - TS$ is the Helmholtz free energy.

System "a" can do useful work until its free energy $F_a = E_a - TS_a$ slides to its minimum. Here

$$0 = \Delta S = \Delta S_a + \Delta S_B$$

$$= \Delta S_a - \frac{\Delta E_a}{T}$$

So

$$0 = -T\Delta S = \Delta F_a = \Delta E_a - T\Delta S_a.$$

Now replace the spring with a rod and push or pull on the piston.

Then

$$\frac{dF_a}{dL} = \frac{d(E_a - TS_a)}{dL} \quad E_a \text{ is constant}$$

$$= -T \frac{dS_a}{dL} = -T \frac{dk \ln V^N}{dL} = -\frac{kNTA}{V}$$

So that

$$-\frac{dF_a}{dL} = \frac{kNT}{L} = f_a.$$

The force of the rod compressing the gas is called an entropic force.

The maximum amount of work system a can do is

$$\int_L^{L_m} f_a dL = -\int_L^{L_m} \frac{dF_a}{dL} dL = F(L) - F(L_m).$$

An abstract definition of pressure p is

$$p = T \left. \frac{dS}{dV} \right|_E, \quad (6.15)$$

We also have

$$T = \left(\left. \frac{dS}{dE} \right|_V \right)^{-1} = \left(\frac{dS}{dQ} \right)^{-1}, \quad (6.9)'$$

16D

Ideal gas:

$$S = k \ln E^{\frac{3N}{2}} V^N$$

$$\left. \frac{dS}{dV} \right|_E = \frac{Nk}{V} = \frac{P}{T}$$

So we again get $pV = NkT$, which is the ideal-gas law.

The Gibbs free energy G is

$$G = E + pV - TS, \quad (6.16)$$

If small system "a" is in thermal and mechanical contact with a huge system B of constant temperature T and constant pressure p , then when we release a constraint on a, new microstates will open until

$$G_a = E_a + pV_a - TS_a$$

is minimized.

The Skinny on Free Energy

First, we define the temperature T as

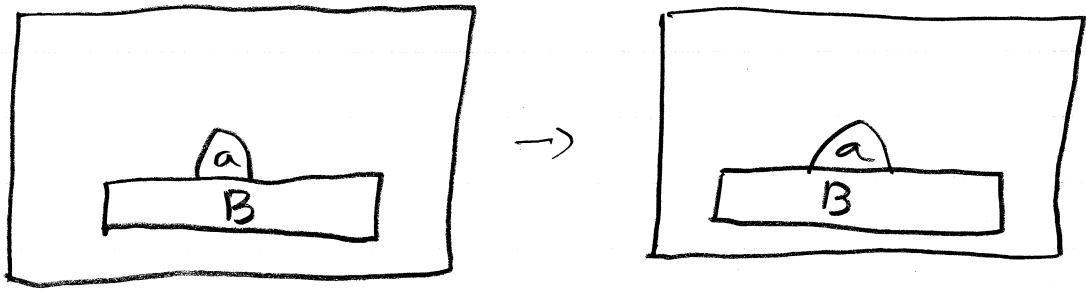
$$T = \left(\frac{\partial S}{\partial E} \Big|_V \right)^{-1} = \left(\frac{\partial S}{\partial \theta} \right)^{-1} \quad (6.9)$$

and the pressure p as

$$p = T \frac{\partial S}{\partial V} \Big|_E \quad (6.15)$$

Second, following Helmholtz, we consider a small system "a" in thermal contact at fixed volume V with a huge thermal reservoir "B." The total entropy $S = S_a + S_B = k \ln \Omega$ will increase to its maximum if we relax some constraints

$$\Delta S = \Delta S_a + \Delta S_B.$$



Heat flows out of a or into a .

$$\Delta E_a = -\Delta E_B.$$

$$\Delta S_B = \left. \frac{\partial S_B}{\partial E_B} \right|_V \Delta E_B = \frac{1}{T} \Delta E_B = -\frac{\Delta E_a}{T}.$$

At equilibrium, $0 = \Delta S$

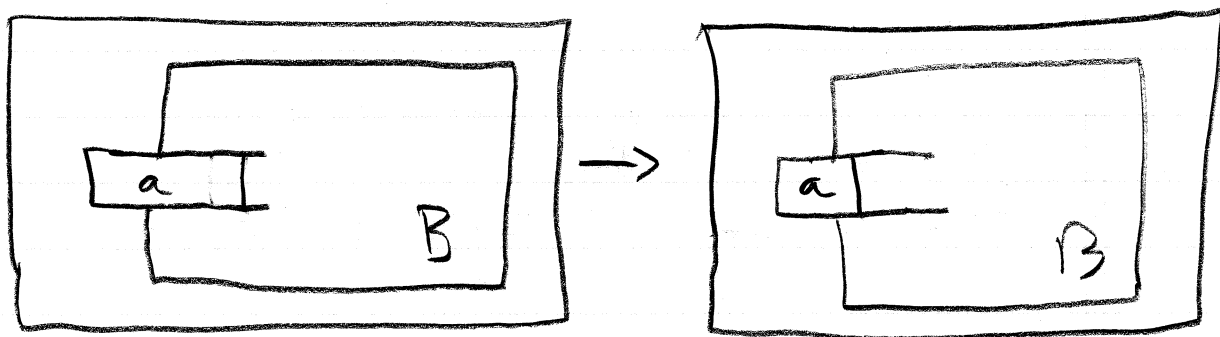
$$0 = \Delta S = \Delta S_a - \frac{\Delta E_a}{T} \quad \text{or}$$

$$0 = \Delta \left(-\frac{F_a}{T} \right) = \Delta S_a - \frac{\Delta E_a}{T}$$

So $F_a = E_a - TS_a$ is a minimum at equilibrium.

Third, following Gibbs, consider a small system "a" in thermal and mechanical contact

with a huge system "B" whose pressure p and temperature T remain constant.



Release constraints and watch the number Ω of microstates grow until $S = S_a + S_B$ is maximal. At its maximum

$$0 = \Delta S = \Delta S_a + \Delta S_B.$$

For small changes of E_B and V_B

$$\frac{1}{T} = \frac{1}{T_B} = \left. \frac{\partial S_B}{\partial E_B} \right|_{V_B} \quad \text{and} \quad \frac{p}{T} = \left. \frac{\partial S_B}{\partial V_B} \right|_E.$$

So

$$0 = \Delta S = \Delta S_a + \frac{\partial S_B}{\partial E_B} \Delta E_B + \frac{\partial S_B}{\partial V_B} \Delta V_B$$

or

$$0 = \Delta S_a + \frac{1}{T} \Delta E_B + \frac{P}{T} \Delta V_B$$

$$0 = \Delta S_a - \frac{\Delta E_a}{T} - \frac{P}{T} \Delta V_a.$$

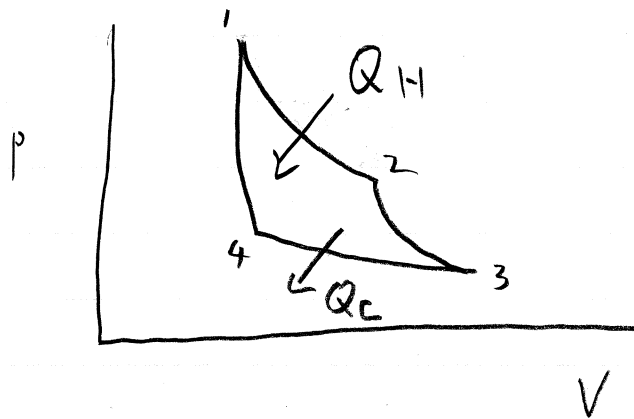
So when S is maximal, the Gibbs free energy

$$G_a = E_a + pV_a - TS_a$$

is minimal because

$$0 = \Delta S_a = \Delta \left(-\frac{G_a}{T} \right).$$

The Carnot cycle



1 → 2 is isothermal at T_H , $Q_H = T_H \Delta S_{12}$

2 → 3 is adiabatic, T_H drops to T_C

3 → 4 is isothermal at T_C , $Q_C = T_C |\Delta S_{34}|$

4 → 1 is adiabatic, T_C rises to T_H

Since $\Delta S_{12} = |\Delta S_{34}|$, the efficiency is

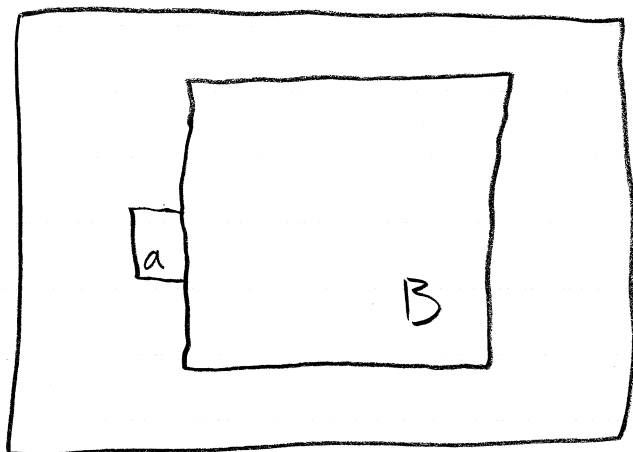
$$e = \frac{W}{Q_H} = \frac{Q_H - Q_C}{Q_H} = 1 - \frac{Q_C}{Q_H} = 1 - \frac{T_C}{T_H}$$

Suppose $T_H = 600^\circ\text{C} \approx 900\text{K}$, if

global warming raises T_C from 300 to 303,

then the efficiency of all such motors

will drop by $\Delta e = -\frac{3}{900} \approx -\frac{1}{3}\%$.



$$E = E_a + E_B$$

We assume that all microstates of total energy E of the joint system $a+B$ are equally likely. Each has probability P_0 . The number of microstates of the huge system B is $\Omega_B(E_B)$.

Since $S_B(E_B) = k \ln \Omega_B(E_B)$,

it follows that

$$\ln \Omega_B(E_B) = S_B(E_B)/k$$

$$\text{or } \Omega_B(E_B) = e^{S_B(E_B)/k}$$

So the probability of a being in a single microstate of energy E_a and B being in any of the $\Omega_B(E_B)$ microstates of energy $E_B = E - E_a$ is

$$P(E_a) = P_0 \Omega_B(E_B) = P_0 e^{S_B(E_B)/k}$$

$$= P_0 e^{S_B(E - E_a)/k}$$

$$\cong P_0 e^{S_B(E) - \frac{E_a}{k} \frac{dS_B(E)}{dE}}$$

But since

$$\frac{1}{T} = \left. \frac{\partial S_B}{\partial E} \right|_V,$$

we have

$$P(E_a) \cong P_0 e^{S_B(E) - \frac{E_a}{kT}}$$

$$\text{or } P(E_a) \cong \text{constant } e^{-\frac{E_a}{kT}}$$

which is the Boltzmann distribution.

If the tiny system a has only two states, then for some constant c

$$P_1 = c e^{-E_1/kT}$$

$$P_2 = c e^{-E_2/kT}$$

and $P_1 + P_2 = 1$. So

$$\frac{P_1}{P_2} = e^{\Delta E/kT} \quad \Delta E = E_2 - E_1$$

So

$$\frac{P_1}{P_2} + 1 = \frac{1}{P_2} = e^{\Delta E/kT} + 1$$

or

$$P_2 = \frac{1}{1 + e^{\Delta E/kT}}$$

and

$$P_1 = \frac{e^{\Delta E/kT}}{1 + e^{\Delta E/kT}} = \frac{1}{1 + e^{-\Delta E/kT}}$$

At high T , $P_1 \approx P_2$. But if $E_2 \gg$

$E_2 \gg E_1 + kT$, then $P_2 \rightarrow 0$ and $P_1 \approx 1$.

6F

Suppose

$$E_a(x, v) = \frac{1}{2} m v^2 + \frac{1}{2} c x^2.$$

Then

$$\langle E_a \rangle = \int_{-\infty}^{\infty} dx \int_{-\infty}^{\infty} dv E_a(x, v) P(x, v)$$

where

$$P(x, v) = N e^{-E_a(x, v)/kT}$$

$$P(x, v) = N e^{-E_a(x, v)/kT}$$

and

$$N = \int_{-\infty}^{\infty} dx \int_{-\infty}^{\infty} dv e^{-E_a(x, v)/kT}$$

So

$$P(x, v) = \left(\sqrt{\frac{m}{2\pi kT}} e^{-\frac{m v^2}{2kT}} \right) \left(\sqrt{\frac{c}{2\pi kT}} e^{-\frac{c x^2}{2kT}} \right)$$

$$\equiv P(v) P(x)$$

in which each factor $P(v)$ & $P(x)$ is itself

normalized. So

$$\langle E_a \rangle = \int dx P(x) \int dv \frac{1}{2} m v^2 P(v)$$

$$+ \int dx \frac{1}{2} c x^2 P(x) \int dv P(v)$$

$$= \int_{-\infty}^{\infty} dv \frac{1}{2} m v^2 P(v) + \int_{-\infty}^{\infty} dx \frac{1}{2} c x^2 P(x).$$

We recall the trick (3.13-3.14)

$$\int_{-\infty}^{\infty} dy y^2 e^{-by^2} = -\frac{d}{db} \int_{-\infty}^{\infty} dy e^{-by^2} = \frac{1}{2} \sqrt{\frac{\pi}{b^3}}.$$

So

$$\int_{-\infty}^{\infty} dv \frac{1}{2} m v^2 P(v) = \int_{-\infty}^{\infty} dv \frac{1}{2} m v^2 \sqrt{\frac{m}{2\pi kT}} e^{-\frac{mv^2}{2kT}}$$

$$= \frac{1}{2} m \sqrt{\frac{m}{2\pi kT}} \int_{-\infty}^{\infty} dv v^2 e^{-\frac{m}{2kT} v^2}$$

$$\left(b = \frac{m}{2kT} \right)$$

$$= \frac{1}{2} m \sqrt{\frac{m}{2\pi kT}} \frac{1}{2} \sqrt{\pi \left(\frac{2kT}{m} \right)^3}$$

$$= \frac{m}{4} \sqrt{\left(\frac{2kT}{m} \right)^2} = \frac{m}{4} \frac{2kT}{m} = \frac{kT}{2}.$$

Similarly,

$$\int_{-\infty}^{\infty} dx \frac{1}{2} m x^2 P(x) = \frac{kT}{2}.$$

$$\text{So } E_a = \frac{1}{2} kT + \frac{1}{2} kT = kT$$

as required by the equipartition of energy.